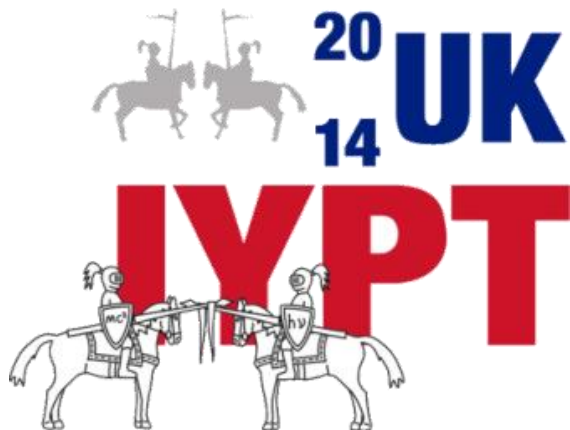


PROBLEM 13.

# ROTATING SADDLE

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Reporter: Domagoj Plušćec

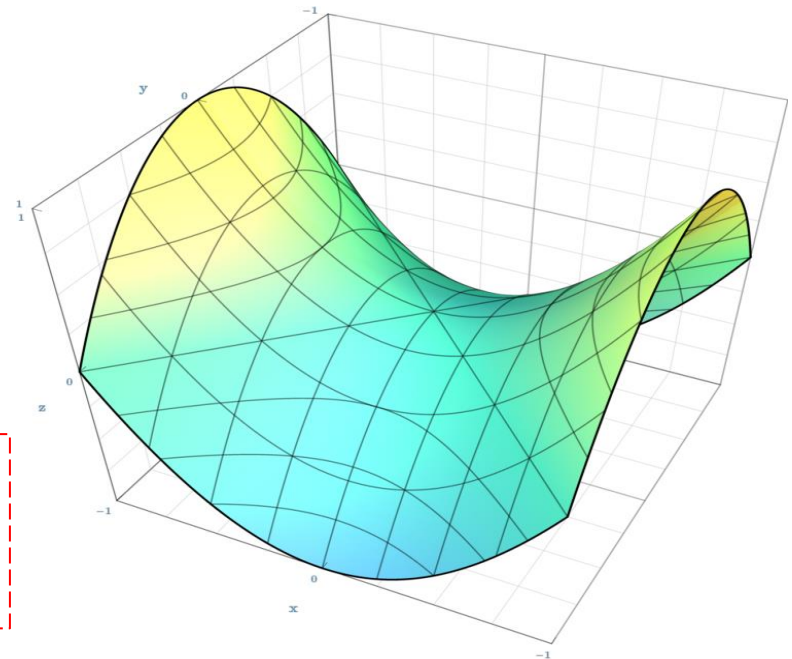


# Problem 13

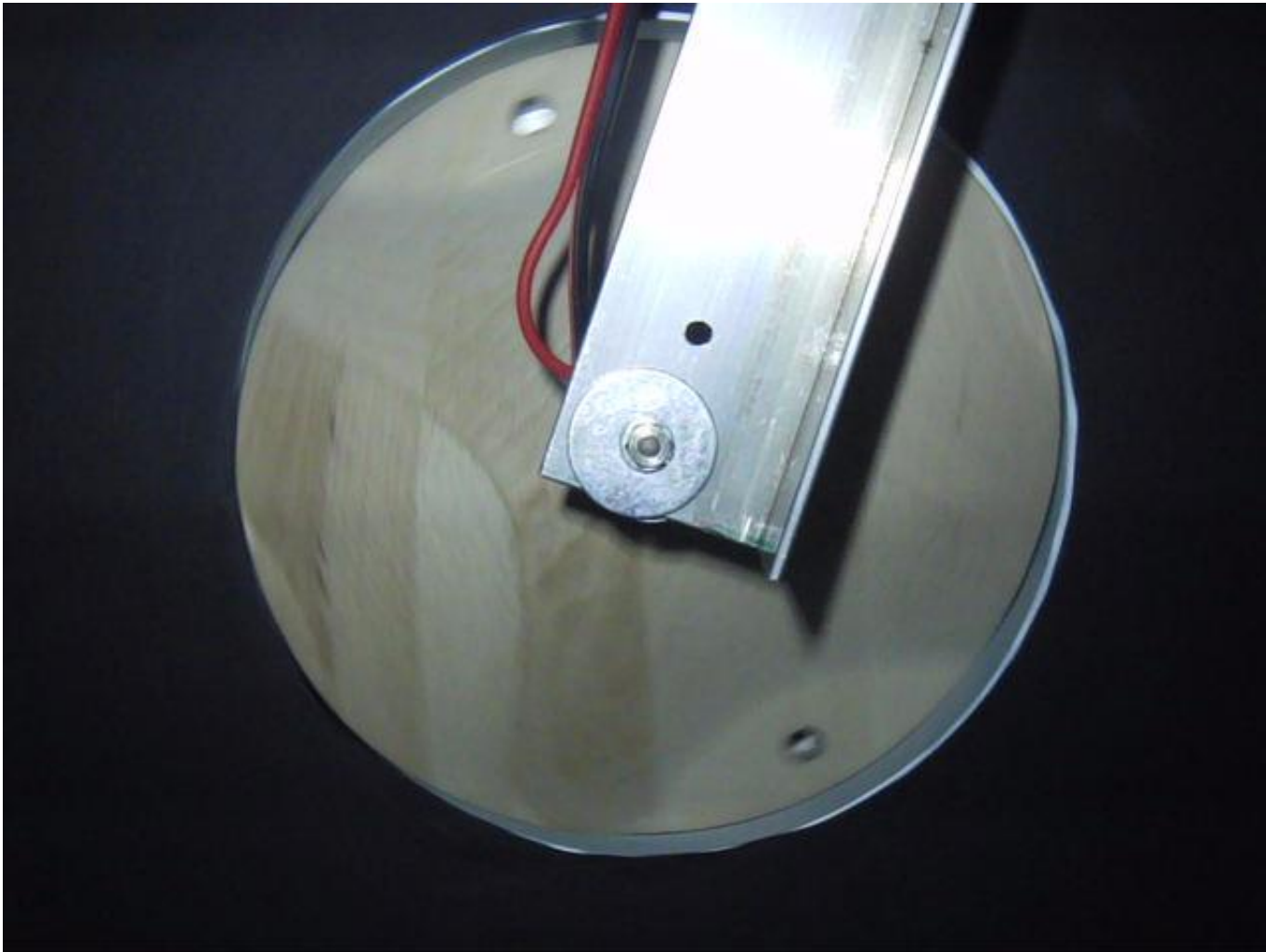
„A ball is placed in the middle of a rotating saddle.  
Investigate its dynamics and explain the conditions under which the ball does not fall off the saddle.”

Saddle - hyperbolic paraboloid

$$z = \frac{x^2}{a} - \frac{y^2}{b}, a, b \in \mathcal{R}^+$$



# Motion example



# Outline

## Theoretical modeling

- Analogy to Paul trap model
- Stability conditions
- Parameters



## Experimental approach

- Saddle construction
- Ball releasing method
- Ball properties
- Determining rotational frequency
- Recording balls motion



## Results

- Ball trajectory
- Dependence of stabilization time on frequency
- Friction influence on stabilization

# Theoretical modeling

- Paul trap analogy (R.I. Thompson, 2002)

Electric field potential  $\longrightarrow$  Gravitational potential

Alternating of electric potential  $\longrightarrow$  Rotation of the saddle

- Forces from rotational frame reference on the ball

$$m\vec{a} = \underbrace{-m\vec{\Omega} \times (\vec{\Omega} \times \vec{r})}_{\text{Centrifugal force}} - \underbrace{2m\vec{\Omega} \times \vec{v}}_{\text{Coriolis force}} - \underbrace{\vec{\nabla}U(*)}_{\text{Force caused by gravitational potential}}$$

$m$  – ball mass

$\Omega$  – saddle angular velocity

$r$  – ball distance from center of the saddle

as analogy we are neglecting friction, because it is difficult to solve analytically and we are going to discuss friction independently

# Gravitational potential of the saddle

$$U = mgz = mg \left( \frac{x^2}{a} - \frac{y^2}{b} \right)$$

$$U = \frac{m}{2} (\omega_1^2 x^2 - \omega_2^2 y^2)$$

Supstitution:

$$\omega_1^2 = \frac{2g}{a}$$

$$\omega_2^2 = \frac{2g}{b}$$

- Assumptions:

- If ball is stabilized it will be near the center of the saddle
- Close to the center of the saddle we neglect the change in z direction ( $\dot{y}, \dot{x} > \dot{z}$ )

- Using assumptions and projecting on (x,y) plane we obtain:

$$\begin{aligned}\ddot{x} - 2\Omega\dot{y} + (\omega_1^2 - \Omega^2)x &= 0 \\ \ddot{y} + 2\Omega\dot{x} - (\omega_2^2 + \Omega^2)y &= 0\end{aligned}$$

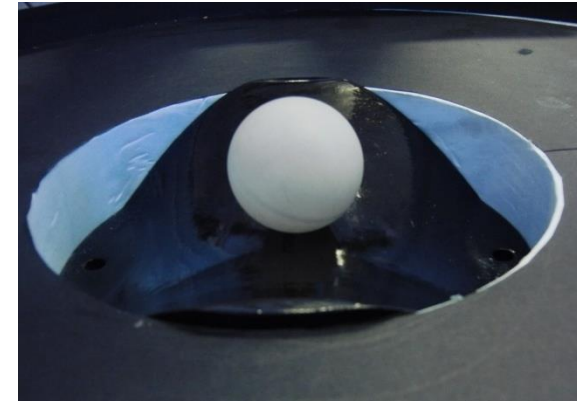
- For equations ansatz is in the form of  $x = c_1 e^{-i\lambda t}$ ,  $y = c_2 e^{-i\lambda t}$

- All solutions for  $\lambda$  must be real for keeping ball stabilized because then solution of the equations is periodic function (Euler formula  $e^{ix} = \cos x + i\sin x$ )
- For lambda we obtain:

$$\lambda_{1,2}^2 = \frac{\omega_2^2 - \omega_1^2 - 2\Omega^2}{2} \pm \frac{1}{2} \sqrt{(\omega_1^2 + \omega_2^2)^2 + 8\Omega^2(\omega_1^2 - \omega_2^2)} > 0$$

# Stability conditions

- $\Omega = 0$  – point  $(0,0)$ 
  - Unstable equilibrium  $\frac{\partial U}{\partial x} = \frac{\partial U}{\partial y} = 0$
  - In x direction stable equilibrium  $\frac{\partial^2 U}{\partial x^2} > 0$
  - In y direction unstable equilibrium  $\frac{\partial^2 U}{\partial y^2} < 0$



- $\Omega > 0$ 
  - $\omega_2^2 \leq \omega_1^2 \leq \Omega^2$

or

  - $\omega_1^2 \leq \omega_2^2 \leq 3\omega_1^2(*)$  and  $\omega_1^2 \leq \Omega^2 \leq \frac{(\omega_1^2 + \omega_2^2)^2}{8(\omega_1^2 - \omega_2^2)}$

Supstitution:

$$\omega_1^2 = \frac{2g}{a}$$

$$\omega_2^2 = \frac{2g}{b}$$

(\*)obtained by investigating saddle curvature properties - Oleg N. Kirillov, 2010



# Parameters



- Saddle shape

- Rotating frequency

- Friction impact

- Moment of inertia of the ball (influence of the rotation of the ball)

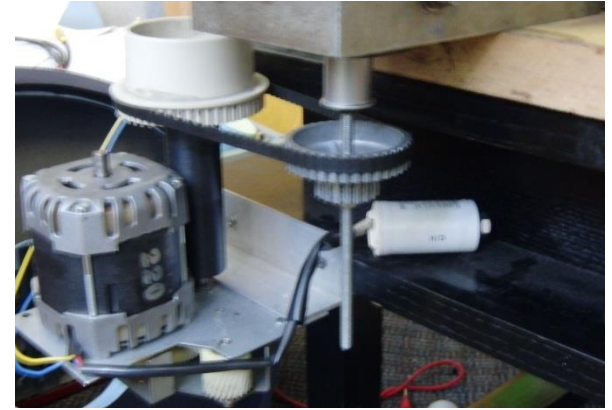
# Experiment

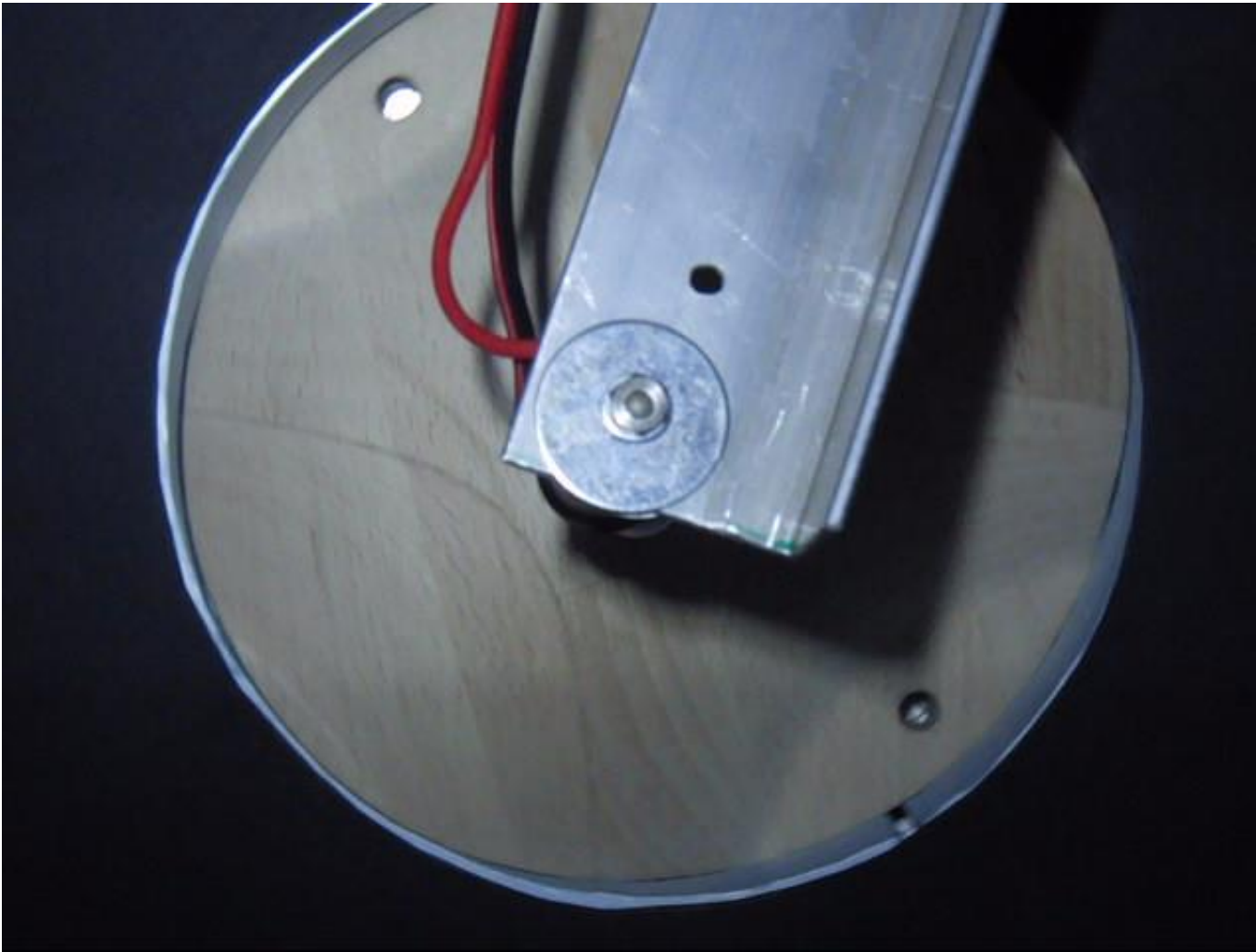
- Saddle construction
  - Saddle made by CNC milling machine
  - Saddle  $z = \frac{x^2}{21} - \frac{y^2}{21}$  [cm], saddle radius 7.5 cm
- Ball releasing
  - Metal balls
    - Released by a coil (interruption of current caused ball drop)
    - Balls with radius 8.5 mm and 12.5 mm
  - Plastic and rubber balls
    - Released by hand
    - Balls with radius: 13 mm, 20 mm, 37.5 mm



# Experiment

- Determining frequency of saddle rotation
  - Determined with stroboscope
  - Frequency of rotation of constructed turntable in range from 1 Hz to 5.5 Hz
- Determining ball motion
  - Motion recorded with high speed camera (120 fps)
  - Tracked in program for video analysis ImageJ
    - Center and axis of symmetry of the saddle were determined by marks on the saddle



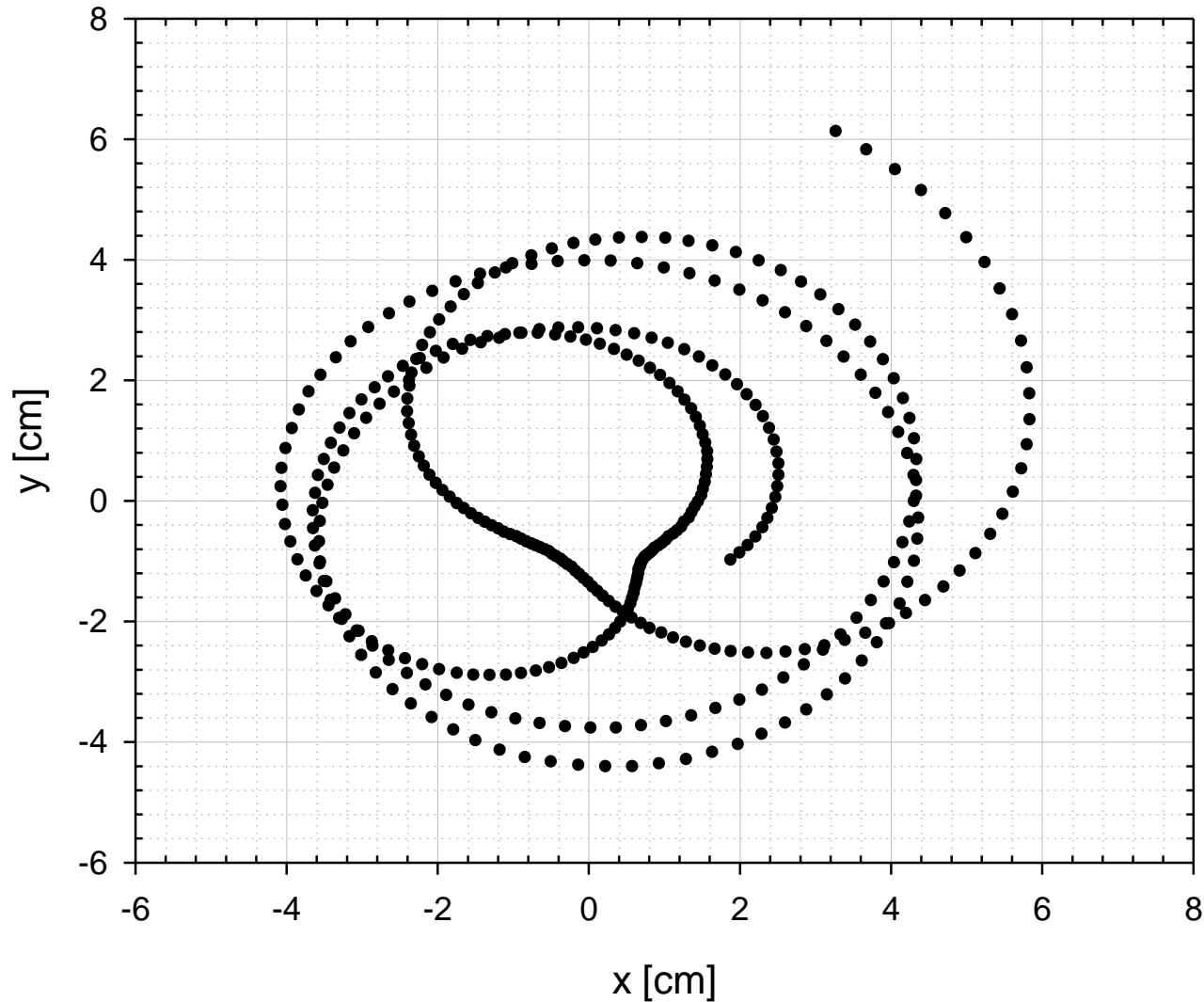


$f = 1.71 \text{ Hz}$

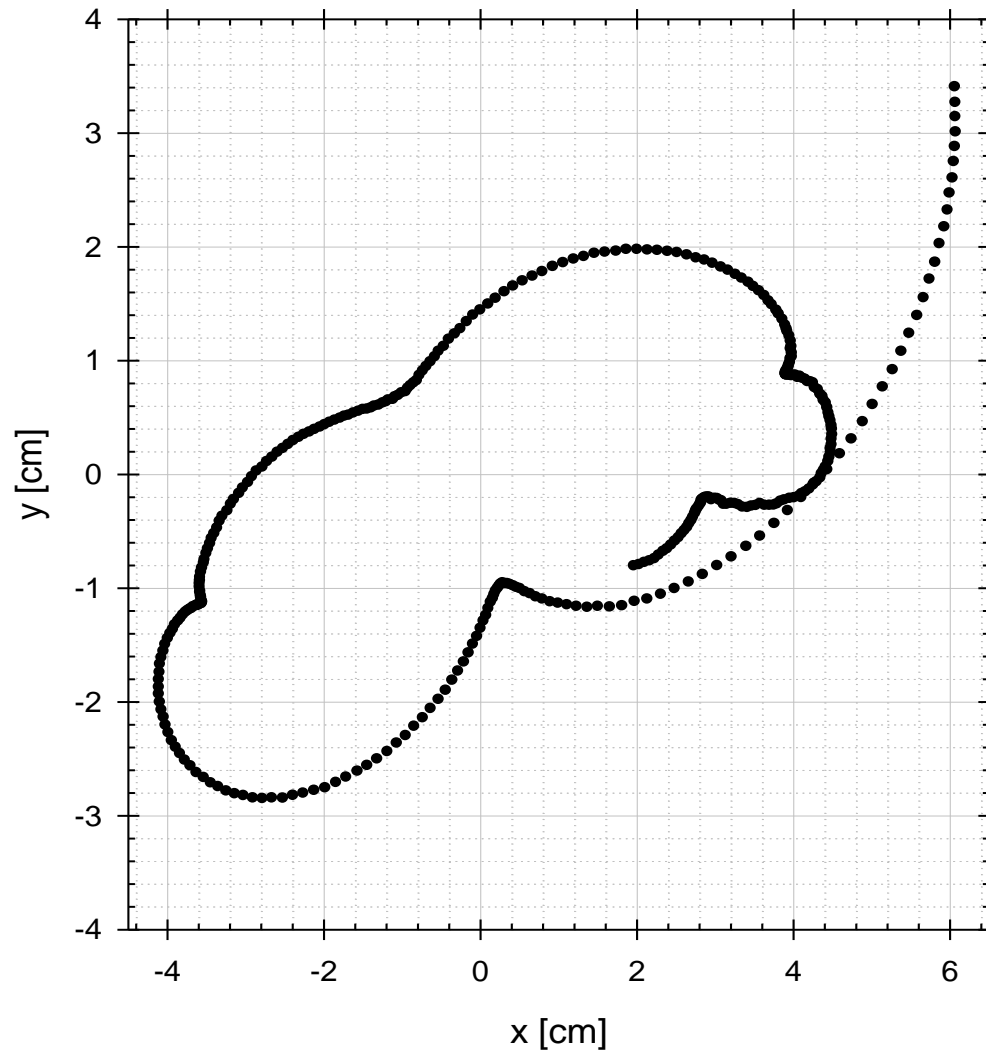
Metal ball

$r = 8.5 \text{ mm}$

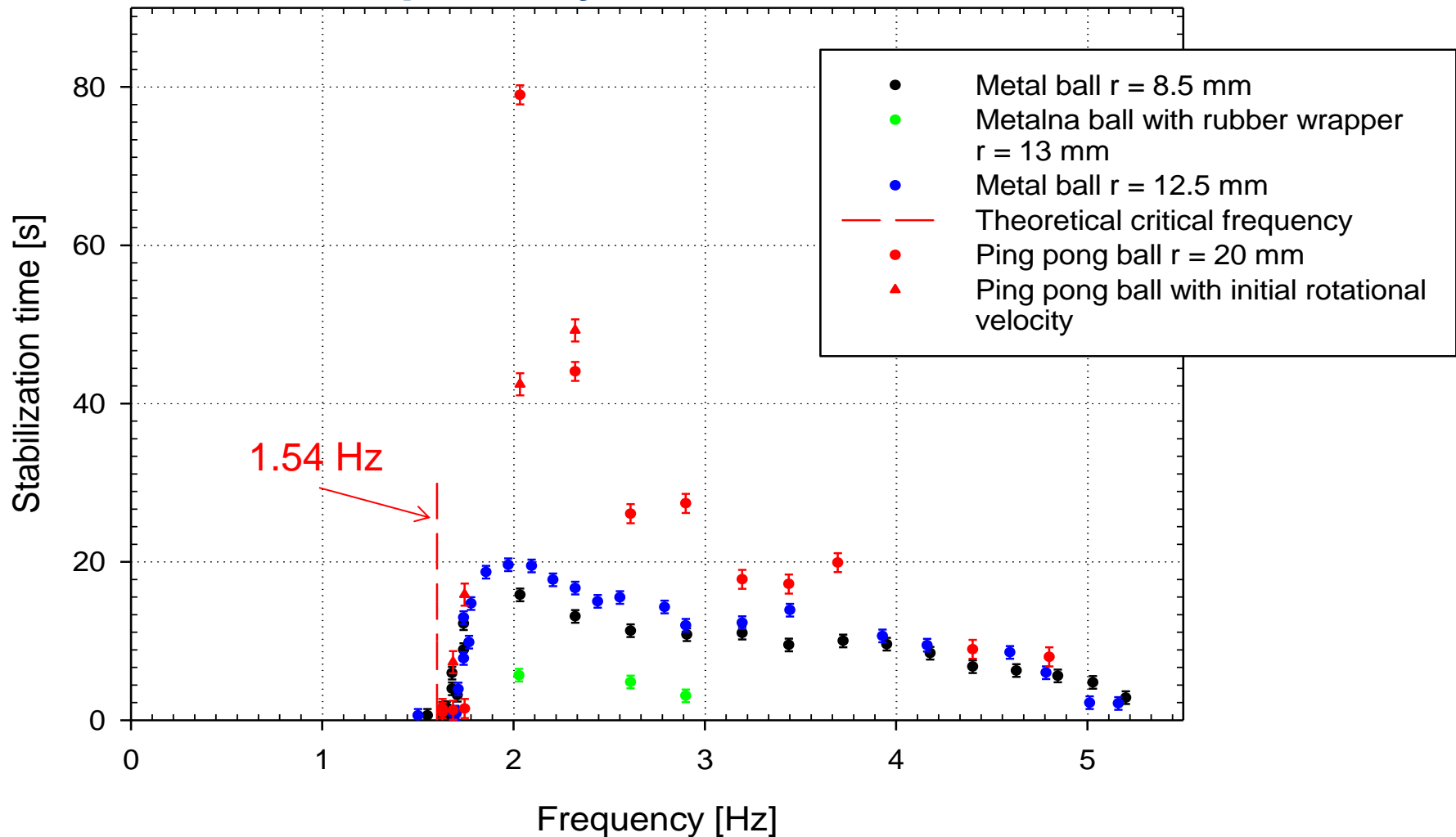
# Ball trajectory - laboratory frame reference



# Ball trajectory - rotational frame reference



# Dependence of stabilization time on saddle frequency

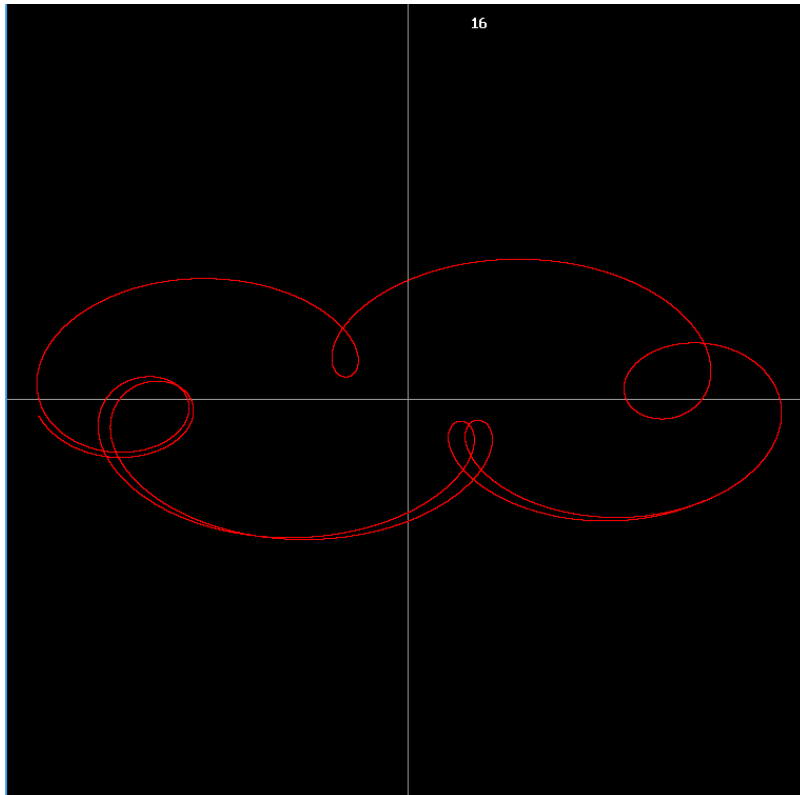


# Friction impact – theoretical analysis

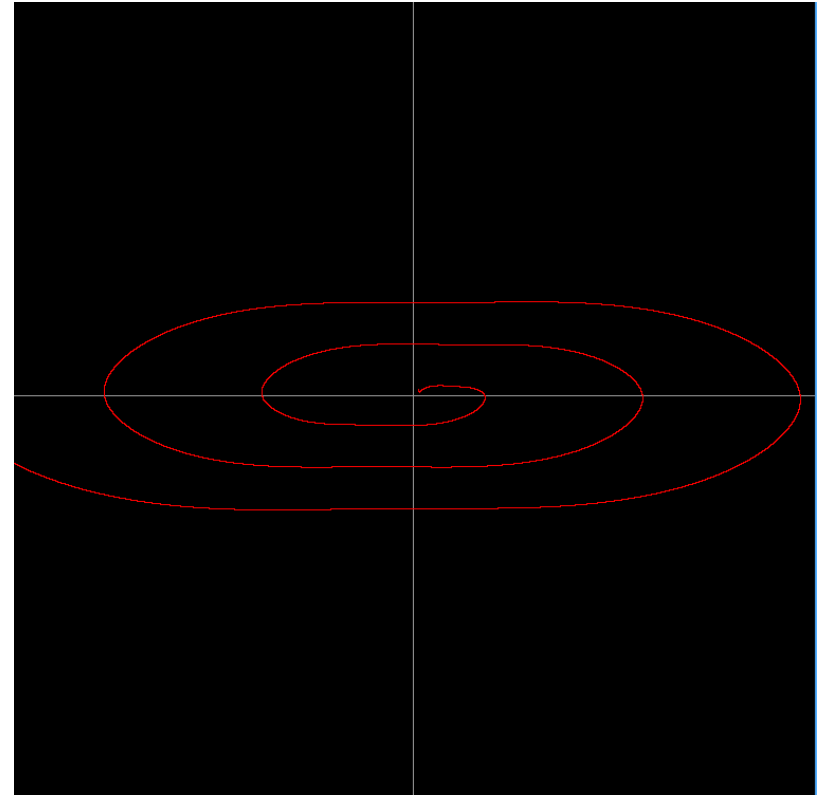
- We assume that when the ball is moving on the saddle that it is rolling
- For ball near the center we neglect it's movement in z direction ( $\dot{y}, \dot{x} > \dot{z}$ )
- We aproximated friction force in forme of  $\vec{F} = \mu m g \frac{\vec{v}}{|v|}$   
where  $\mu$  is coefficient of rolling friction
- $\mu \sim 0.001^*$
- We solved numericaly equations from our model and numericaly added friction force and observed how it influences our model

\*Physics and Chemistry of Interfaces; Hans-Jürgen Butt, Karlheinz Graf, 2003





$\mu = 0; f = 1.6 \text{ Hz}$   
picture dimensions 0.9x0.9 cm



$\mu = 0.005; f = 1.6 \text{ Hz}$   
picture dimensions 15x15 cm

# Conclusion

- We constructed a saddle and experimental setup for rotation of a saddle and system for releasing metal balls
- Simple theoretical model for point mass
  - Stabilization conditions:
    - $\omega_2^2 \leq \omega_1^2 \leq \Omega^2$  or
    - $\omega_1^2 \leq \omega_2^2 \leq 3\omega_1^2$  and  $\omega_1^2 \leq \Omega^2 \leq \frac{(\omega_1^2 + \omega_2^2)^2}{8(\omega_1^2 - \omega_2^2)}$
  - Assumption:
    - $\dot{y}, \dot{x} > \dot{z}$  for the ball near the center of the saddle
- We discussed friction impact on ball stability
  - Friction will always destabilises ball on the rotating saddle

# Literatura

1. Johan Nilsson; Trapping massless Dirac particles in a rotating saddle; Phys. Rev. Lett. 111, 100403 (2013)
2. R.I.Thompson, T.J. Harmon, M.G. Ball; The rotating saddle trap: a mechanical analogy to RF-electric-quadrupole ion trapping?; Can. J. Phys. 80, 1433-1448 (2002)
3. Oleg N. Kirillov, Brouwer's problem on a heavy particle in a rotating vessel: wave propagation, ion traps, and rotor dynamics; Physics Letters A 375, 1653-1660 (2011)
4. Physics and Chemistry of Interfaces. Hans-Jürgen Butt, Karlheinz Graf, Michael Kappl, 2003

# Stroboscopic picture of the balls motion



IYPT 2014  
CROATIAN TEAM

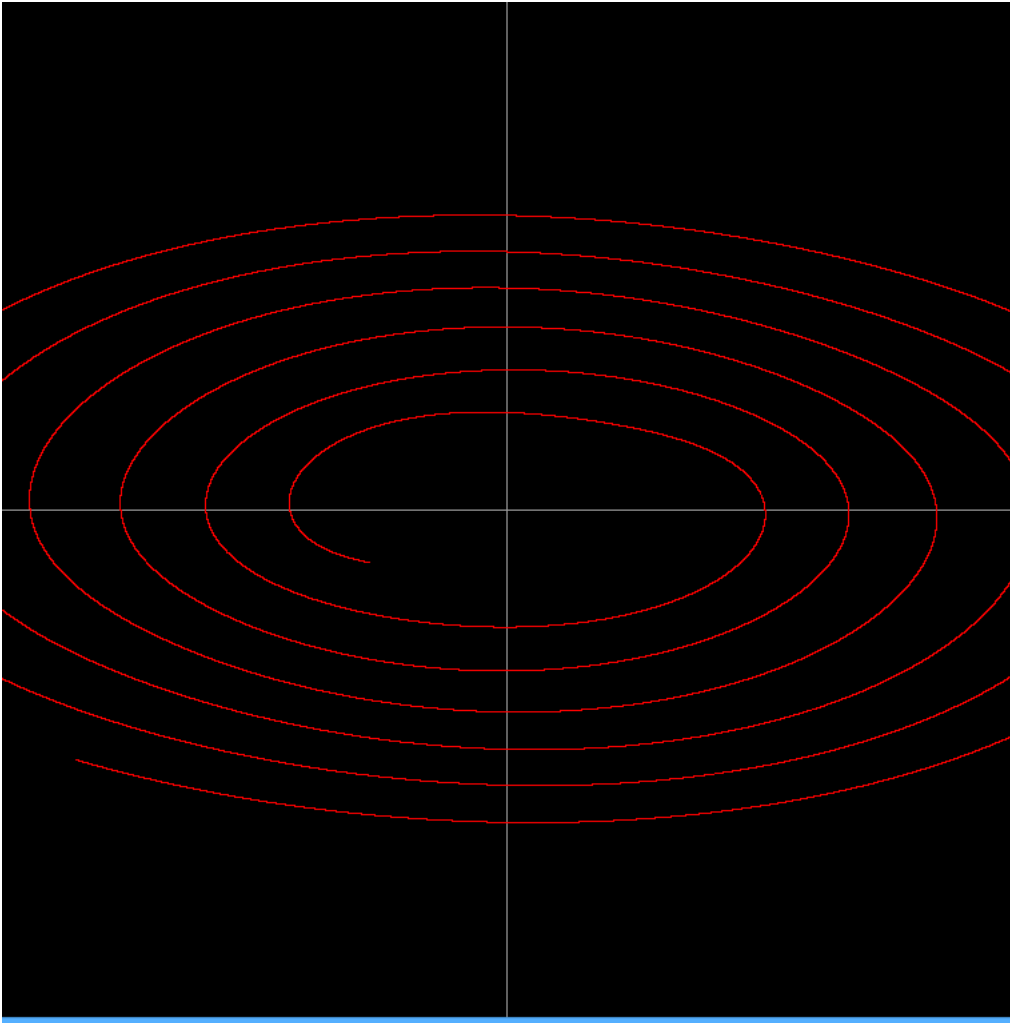
# THANK YOU

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Reporter: Domagoj Plušćec

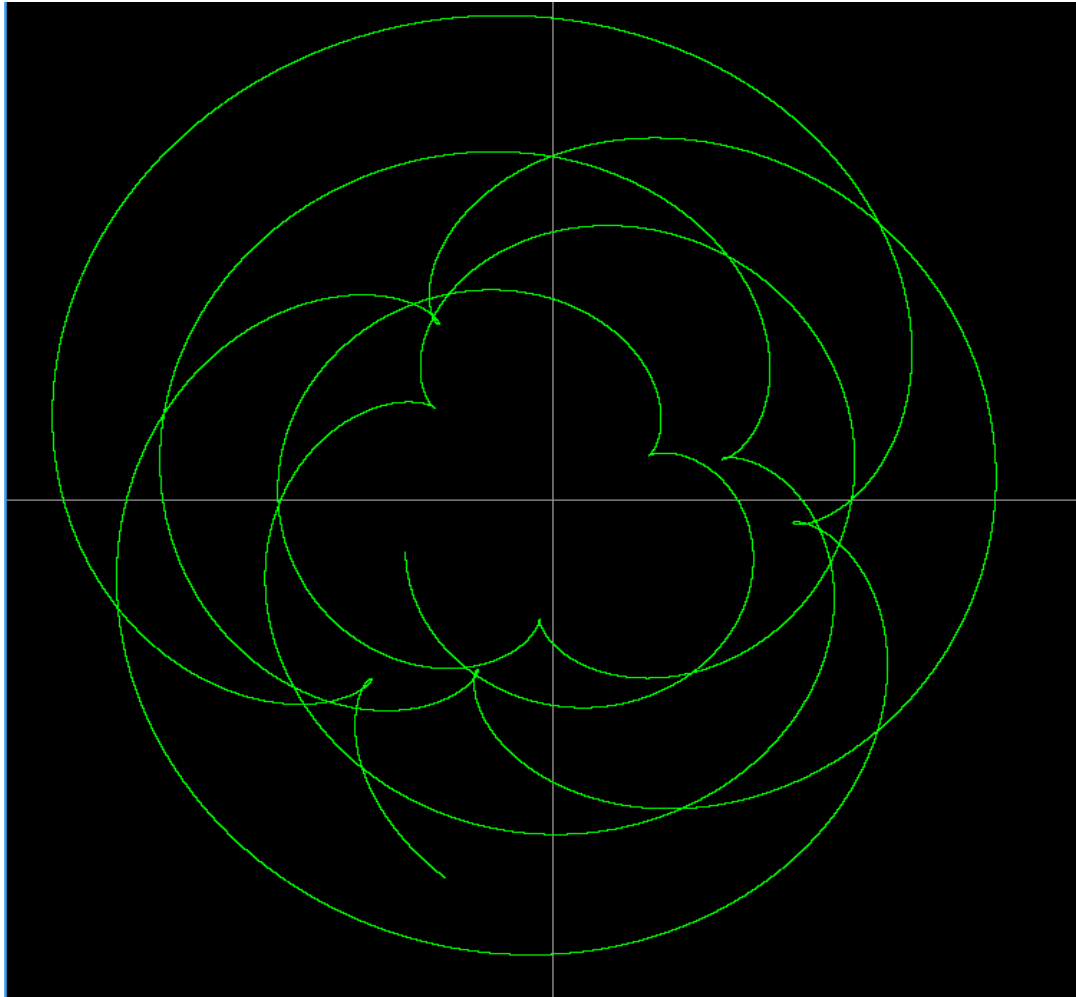


# Ball trajectory – rotational frame reference

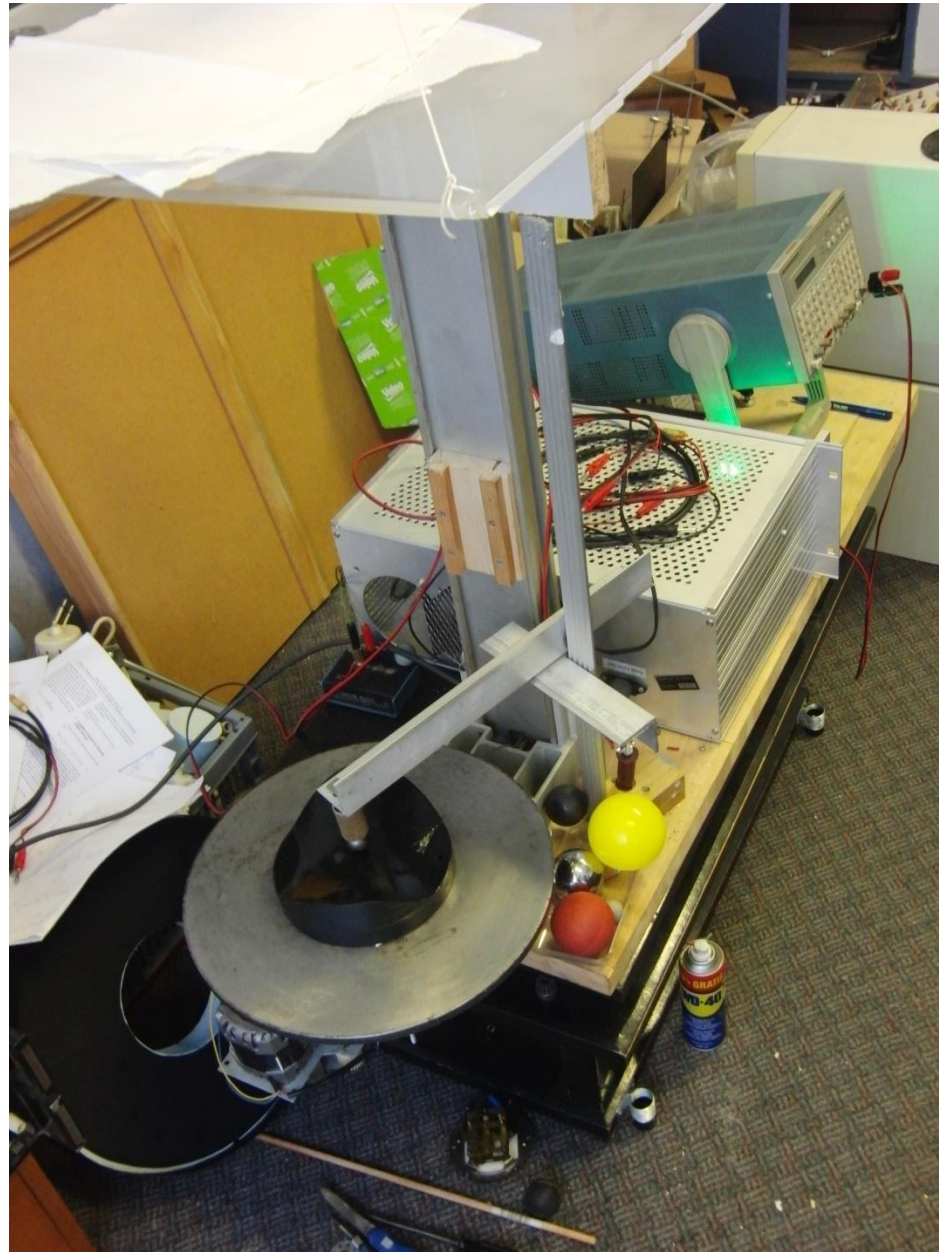


$$\mu = 0.005; f = 1.71 \text{ Hz}$$

# Ball trajectory - laboratory frame reference



$$\mu = 0.005; f = 1.71 \text{ Hz}$$





$$\begin{aligned}\ddot{x} - 2\Omega\dot{y} + (\omega_1^2 - \Omega^2)x &= 0 \\ \ddot{y} + 2\Omega\dot{x} - (\omega_2^2 + \Omega^2)y &= 0\end{aligned}$$

$$\begin{aligned}-\lambda^2 c_1 + 2c_2 \lambda \Omega i + (\omega_1^2 - \Omega^2)c_2 &= 0 \\ \lambda^2 c_2 + 2c_1 \lambda \Omega i + (\omega_2^2 + \Omega^2)c_2 &= 0\end{aligned}$$

*Det=0*

$$\lambda^4 + (\omega_2^2 - \omega_1^2 - 2\Omega^2)\lambda^2 - (\omega_1^2 - \Omega^2)(\omega_2^2 + \Omega^2) = 0$$

$$\lambda_{1,2}^2 = \frac{\omega_2^2 - \omega_1^2 - 2\Omega^2}{2} \pm \frac{1}{2} \sqrt{(\omega_1^2 + \omega_2^2)^2 + 8\Omega^2(\omega_1^2 - \omega_2^2)}$$